## Straight Aluminum Fin EXAMPLE 2-9

An aluminum fin  $[k = 200 \text{ W/m} \cdot ^{\circ}\text{C}] 3.0 \text{ mm}$  thick and 7.5 cm long protrudes from a wall, as in Figure 2-9. The base is maintained at 300°C, and the ambient temperature is 50°C with  $h = 10 \text{ W/m}^2 \cdot ^{\circ}\text{C}$ . Calculate the heat loss from the fin per unit depth of material.

#### **Solution:**

Let neglecting the heat lost from the end

$$q = \sqrt{hPkA} \theta_0 \tanh mL$$
  

$$P = 2(W + t) = 2(1 + 0.003) = 2.006m$$
  

$$A = W.t = 1 \times 0.003 = 0.003m^2$$
  

$$\theta_0 = T_0 - T_\infty = 300 - 50 = 250^{\circ}C$$
  

$$m = \sqrt{\frac{hP}{kA}} = \sqrt{\frac{10 \times 2.006}{200 \times 0.003}} \approx 5.774$$
  

$$\therefore$$
  

$$q = \sqrt{10 \times 2.006 \times 200 \times 0.003} \times 250 \times \tanh(5.774 \times 0.075)$$
  

$$q = 357 W/m depth$$

**Example:** An experimental device that produce excess heat is supplied with pin fins to increase the rate of cooling consider a copper pin fin, 0.25cm in diameter that produces from a wall at 95°C into ambient air at 25°C.  $h=10W/m^2$ .K, k=396 W/mK. Calculate the heat loss assuming that:

- 1- The fin is infinitely long.
- 2- The fin is 2.5cm long and heat is convected from the end.

 $\cosh mL + (h/mk) \sinh mL$ 

3- How long would the fin have to be for the infinitely long solution to be corrected within 5%.

### Solution:

1)  

$$q = \sqrt{hPkA} \theta_0$$
  
 $P = \pi.D, \quad A = \frac{\pi.D^2}{4}$   
 $q = \sqrt{10 \times \pi \times 0.0025 \times 396 \times \pi / 4 \times (0.0025)^2} \times (95 - 25) = 0.865W$   
2)  
 $q = \sqrt{hPkA} \theta_0 \frac{\sinh mL + (h/mk) \cosh mL}{h^2} = 0.14W$ 

3) For only 5% error  $\frac{\sinh mL + (h/mk) \cosh mL}{\cosh mL + (h/mk) \sinh mL} \ge 0.95$ 

Which gives L≥28.3cm

# **Fin Effectiveness**

The effectiveness of the fin in transferring heat is given by the *fin efficiency*  $(\eta_f)$ .

 $\eta_{f} = \frac{\text{actual heat transferred}}{\text{heat that would be transferred if entire fin area were at base temperature}} = \frac{q_{fin}}{q_{\text{max}}}$  $q_{\text{max}} = h.A_{f}.\theta_{0}, \quad A_{f} = P.L, A_{f} = surface area of the fin$ 

For case 2:  $q_{fin} = \sqrt{hPkA} \theta_0 \tanh mL \implies$  $\eta_f = \frac{\sqrt{hPkA} \theta_0 \tanh mL}{hPL\theta_0} = \frac{\tanh mL}{mL}$ 

# **Fin Performance (E)**

$$\varepsilon = \frac{q \text{ with fin}}{q \text{ without fin}} = \frac{\eta_f A_f h \theta_0}{h A_b \theta_0}$$
$$A_f = \text{surface area of the fin} = P.L$$
$$A_b = \text{base area of the fin} = A$$

# **Corrected Length (Lc)**

Lc is used in all equations that apply for the case of the fin with an insulated tip (case 2).

$$Lc = L + \frac{t}{2}$$
 (For general)  
 $Am = L.t$ 

**Example:** The outer surface of an oil heater at a uniform temperature of  $150^{\circ}$ C is to be filled with straight rectangular fins having a uniform thermal conductivity of 25 W/m.K. The ambient air temperature is 20°C and the heat transfer coefficient is 570 W/m<sup>2</sup>.K. Determine the thickness and fin efficiency if the length of each fin is 10mm and each to remove 900W per meter length of primary surface.

#### <u>Solution.</u>

$$T_0 = 150^{\circ}\text{C}, \quad k = 25W/m.K, \quad h = 570W/m^2.K, \quad L = 10mm, \quad T_{\infty} = 20^{\circ}\text{C}$$

$$q = \sqrt{hPkA\theta_0} \tanh mLc$$

$$P = 2(w+t) = 2(t+1), \quad A = wt, \quad w = 1$$

$$m = \sqrt{\frac{hP}{kA}} = \sqrt{\frac{570 \times 2(t+1)}{25 \times t}} = 6.75 \sqrt{\frac{t+1}{t}}$$

$$q = \sqrt{570 \times 2(t+1) \times 25 \times t} (150 - 20) \tanh\left(0.0675 \sqrt{\frac{t+1}{t}}\right)$$

$$900 = 21964.5 \sqrt{t^2 + t} \tanh\left(0.0675 \sqrt{\frac{t+1}{t}}\right)$$
by trial and error  $\Rightarrow t \approx 2.07mm$ 

$$\eta_f = \frac{\tanh mLc}{mLc} = 60.7\%$$

**Example 2:** A very long 1cm diameter copper rod (k = 377 W/m.K) is exposed to an environment at 22°C. The base temperature is maintained at 150°C and the heat transfer coefficient between the rod and surrounding air is 11 W/m<sup>2</sup>.K. Determine the heat transfer rate from the rod to the surrounding air.

#### Solution.

$$q = \sqrt{hPkA}\theta_0$$
  

$$q = \sqrt{h(\pi D) \times k \times \pi D^2 / 4} (T_0 - T_\infty)$$
  

$$q = 12.95W$$

**Example 3:** Repeat example 2 for finite length of 2, 4, 8, .....128cm, a surrounding heat loss at the end (case 3).

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### <u>Solution.</u>

For case 3

$$q = \sqrt{hPkA} \theta_0 \frac{\sinh mL + (h/mk) \cosh mL}{\cosh mL + (h/mk) \sinh mL}$$

$$m = \sqrt{\frac{hP}{kA}} = \sqrt{\frac{h(\pi D)}{k(\frac{\pi D^2}{4})}} = \sqrt{\frac{4h}{kD}} = 3.416$$

At L=2cm, mL=0.06832, sinhmL=0.06837, coshmL=1.00233  $\Rightarrow$  q =0.993W