



Class: 2<sup>nd</sup>

Subject: Strength of Materials

Lecturer: M.Sc Murtadha Mohsen Al-Masoudy

E-mail: [Murtadha\\_Almasoody@mustaqbal-college.edu.iq](mailto:Murtadha_Almasoody@mustaqbal-college.edu.iq)



*Al-Mustaqbal University College*  
*Air Conditioning and Refrigeration Techniques*  
*Engineering Department*

**Strength of Materials**

**Second Stage**

**M.Sc Murtadha Mohsen Al-Masoudy**

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م.س. مورتادها موهسن الماسودي

# Simple Strains

## NORMAL STRAIN UNDER AXIAL LOADING

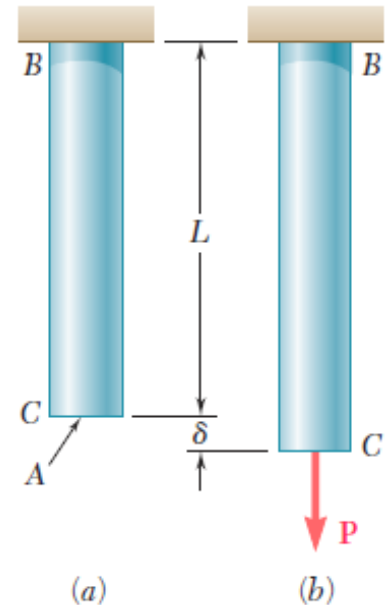
### Concepts of Strain:

- ▶ As already mentioned, wherever a single force (or a system of forces) acts on a body, it undergoes some deformation (Figure 6.1). This deformation per unit length is known as strain. Mathematically strain may be define as deformation per unit length, i.e., strain is:

$$\epsilon = \frac{\delta l}{l} \quad \text{or} \quad \delta l = \epsilon \cdot l$$

Where  $\delta l$  =Change of length of the body, and

$l$  =Original length of the body

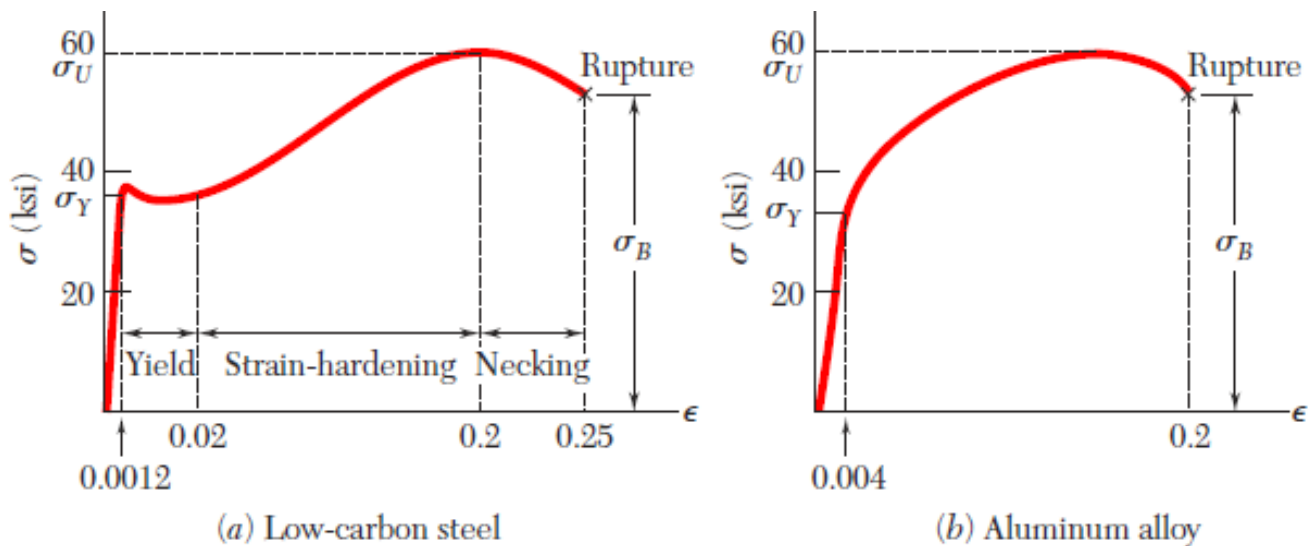


**Figure 6.1**

- ▶ Strain is thus, a measure of the deformation of the material and is a non-dimensional quantity i.e. it has no units. It is simply a ratio of two quantities with the same unit.

### Stress – Strain Relationship

- ▶ When a material is loaded, the stress is proportional to the strain mathematically, and as shown in Figure (6.2),



**Figure (6.2): Stress-strain diagrams of two typical ductile materials**

- ▶ The initial portion of the stress – strain diagram for most material used in engineering structures is a straight line. For the initial portion of the diagram, the stress ( $\sigma$ ) is directly proportional to the strain ( $\epsilon$ ). Therefore, for a specimen subjected to a uniaxial load, can write:

$$\frac{\text{Stress}}{\text{Strain}} = E \rightarrow \sigma = E\epsilon$$

This relationship is known as *Hook's Law*.

- ❖ **Note:** *Hook's Law* describes only the initial linear portion of the stress – strain curve for a bar subjected to uniaxial extension.
- ❖ The slope of the straight line portion of the stress – strain diagram is called the *Modulus of Elasticity* or *Young's Modulus*.

$$E = \frac{\sigma}{\epsilon}$$

### Deformation of a Body due to Force acting on it

Consider a body subjected to a tensile force as shown in Figure (6.3).

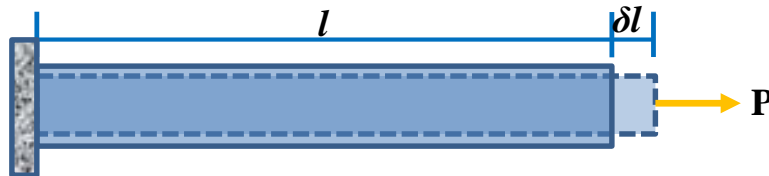


Figure (6.3)

Let  $P$  =Load or force acting on the body,

$l$  =Length of the body,

$A$  =Cross – sectional area of the body,

$\sigma$  =Stress induced in the body,

$E$  =Modulus of elasticity for the material of the body,

$\epsilon$  =Strain, and

$\delta l$  =Deformation of the body.

Knowing that the stress is:

$$\sigma = \frac{P}{A}$$

And the strain is:

$$\varepsilon = \frac{\sigma}{E} = \frac{\frac{P}{A}}{E} = \frac{P}{EA}$$

Then the deformation is:

$$\delta l = \varepsilon(l) = \frac{Pl}{EA}$$

### Examples

**Example (6.1):** A steel rod (1 m) long and (20 mmx20mm) in cross – section is subjected to a tensile force of (40 kN). Determine the elongation of the rod, if modulus of elasticity for the rod material is (200 GPa).

**Solution:**

$$\delta l = \frac{Pl}{EA} = \frac{40 \times 10^3 (1 \times 10^3)}{200 \times 10^3 (20 \times 20)} = 0.5 \text{ mm}$$

**Example (6.2):** In an experiment, a steel specimen of (13mm) diameter was found to elongate (0.2 mm) in a (200 mm) gauge length when it was subjected to a tensile force of (26.8 kN). If the specimen was tested within the elastic range, what is the value of Young’s modulus for the steel specimen?

**Solution:**

$$\delta l = \frac{Pl}{EA} \rightarrow 0.2 = \frac{26.8 \times 10^3 (200)}{E \left( \frac{\pi}{4} (13)^2 \right)}$$

$$E = 201910.2 \text{ MPa} \text{ or } 201.91 \text{ GPa}$$

**Example (6.3):** Determine the deformation of the steel rod shown in Figure (6.4) under the given loads.  $E=29 \times 10^6$  psi.

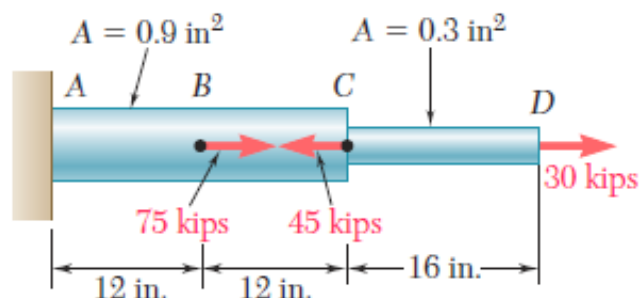


Figure (6.4)

**Solution:**

- We divide the rod into three component parts shown in Figure (6.5b) and write:

$$L_1 = L_2 = 12 \text{ in.}$$

$$L_3 = 16 \text{ in.}$$

$$A_1 = A_2 = 0.9 \text{ in}^2$$

$$A_3 = 0.3 \text{ in}^2$$

Section in each part as shown in Figure (6.5c), then:

$$\sum F_x = 0$$

$$P_1 = 60 \text{ kips} = 60 \times 10^3 \text{ lb}$$

$$P_2 = -15 \text{ kips} = -15 \times 10^3 \text{ lb}$$

$$P_3 = 30 \text{ kips} = 30 \times 10^3 \text{ lb}$$

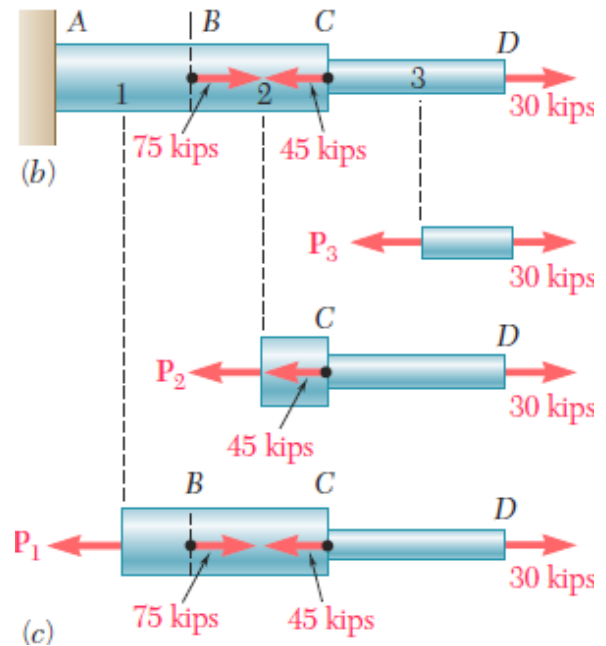
The total deformation in steel bar is:

$$\delta L = \sum \frac{P_i L_i}{E_i A_i}$$

$$\delta L = \frac{1}{29 \times 10^6} \left( \frac{60 \times 10^3 (12)}{0.9} + \frac{-15 \times 10^3 (12)}{0.9} + \frac{30 \times 10^3 (16)}{0.3} \right) = \frac{2.2 \times 10^6}{29 \times 10^6} = 75.9 \times 10^{-3} \text{ in.}$$

**Example (6.4):** The rigid bar *BDE* is supported by two links *AB* and *CD*. Link *AB* is made of aluminum ( $E=70 \text{ GPa}$ ) and has a cross-sectional area of  $500 \text{ mm}^2$ ; link *CD* is made of steel ( $E=200 \text{ GPa}$ ) and has a cross-sectional area of  $600 \text{ mm}^2$ . For the 30-kN force shown in Figure (6.6), determine the deflection

- (a) of *B*,
- (b) of *D*,
- (c) of *E*.



**Figure (5.5)**

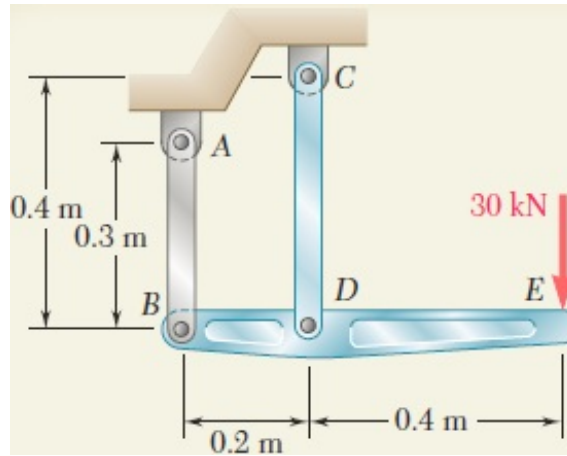


Figure (6.6)

**Solution:**

**Bar BDE as F.B.D:**

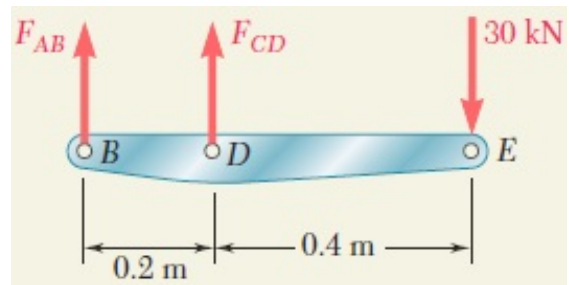
$$\circlearrowleft \sum M_B = 0$$

$$F_{CD}(0.2) - 30(0.6) = 0$$

$$F_{CD} = +90 \text{ kN (Tens.)}$$

$$\uparrow \sum F_y = 0 \rightarrow F_{AB} + F_{CD} - 30 = 0$$

$$\rightarrow F_{AB} = 30 - 90 = -60 \text{ (Comp.)}$$



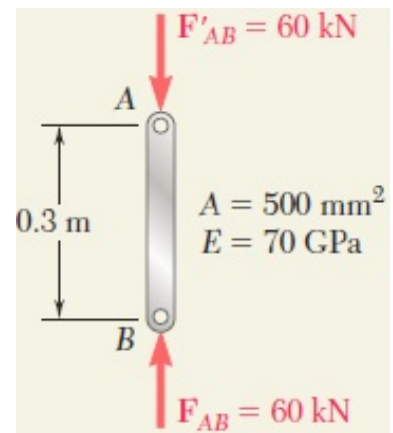
**(a) Deflection of B:**

$$\delta_B = \frac{Pl}{EA} = \frac{(-60 \times 10^3)(0.3)}{(70 \times 10^9)(500 \times 10^{-6})}$$

$$\delta_B = -514 \times 10^{-6} \text{ m or } \delta_B = -0.514 \text{ mm}$$

The negative sign indicates a contraction of member AB, and, thus, an upward deflection of end B:

$$\delta_B = 0.514 \text{ mm } \uparrow$$

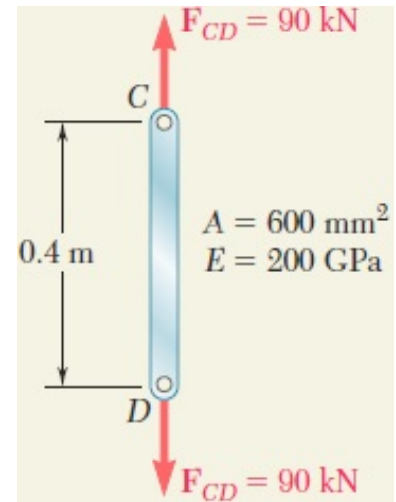


**(b) Deflection of D:**

$$\delta_D = \frac{Pl}{EA} = \frac{(90 \times 10^3)(0.4)}{(200 \times 10^9)(600 \times 10^{-6})}$$

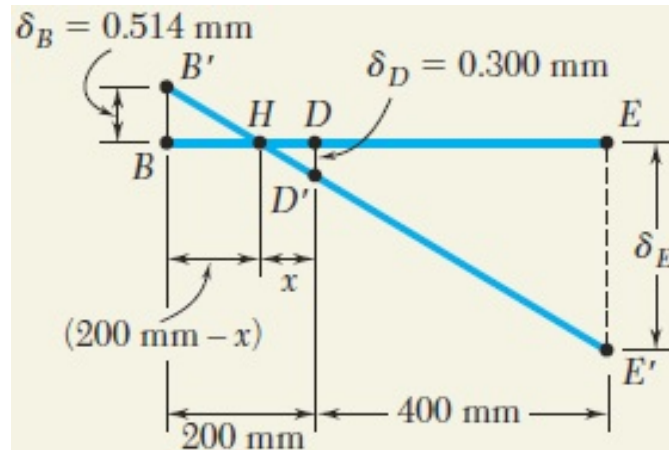
$$\delta_D = 300 \times 10^{-6} \text{ m or}$$

$$\delta_D = 0.300 \text{ mm} \downarrow$$



**(c) Deflection of E:**

We denote by  $B'$  and  $D'$  the displaced positions of points  $B$  and  $D$ . Since the bar  $BDE$  is rigid, points  $B'$ ,  $D'$ , and  $E'$  lie in a straight line and we write:



$$\frac{BB'}{DD'} = \frac{BH}{HD} \rightarrow \frac{0.514}{0.300} = \frac{200 - x}{x} \rightarrow x = 73.7 \text{ mm}$$

$$\frac{EE'}{DD'} = \frac{HE}{HD} \rightarrow$$

$$\frac{\delta_E}{73.7} = \frac{400 + 73.7}{73.7}$$

$$\rightarrow \delta_E = 1.928 \text{ mm} \downarrow$$

*Murtadha Al-Masoudy*  
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