



Class: 2nd

Subject: Strength of Materials

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Strength of Materials

Second Stage

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Simple Strains

NORMAL STRAIN UNDER AXIAL LOADING

Concepts of Strain:

- ▶ As already mentioned, wherever a single force (or a system of forces) acts on a body, it undergoes some deformation (Figure 6.1). This deformation per unit length is known as strain. Mathematically strain may be define as deformation per unit length, i.e., strain is:

$$\epsilon = \frac{\delta l}{l} \quad \text{or} \quad \delta l = \epsilon \cdot l$$

Where δl =Change of length of the body, and

l =Original length of the body

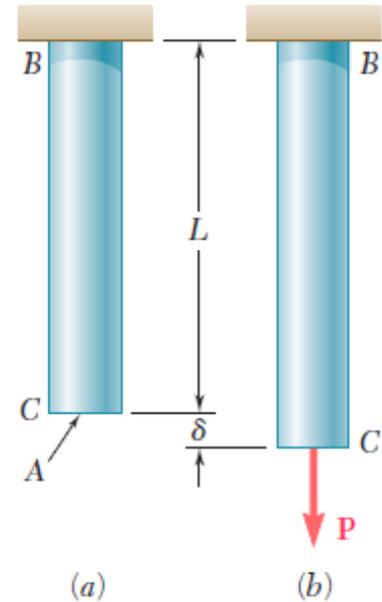


Figure 6.1

- ▶ Strain is thus, a measure of the deformation of the material and is a non-dimensional quantity i.e. it has no units. It is simply a ratio of two quantities with the same unit.

Stress – Strain Relationship

- ▶ When a material is loaded, the stress is proportional to the strain mathematically, and as shown in Figure (6.2),

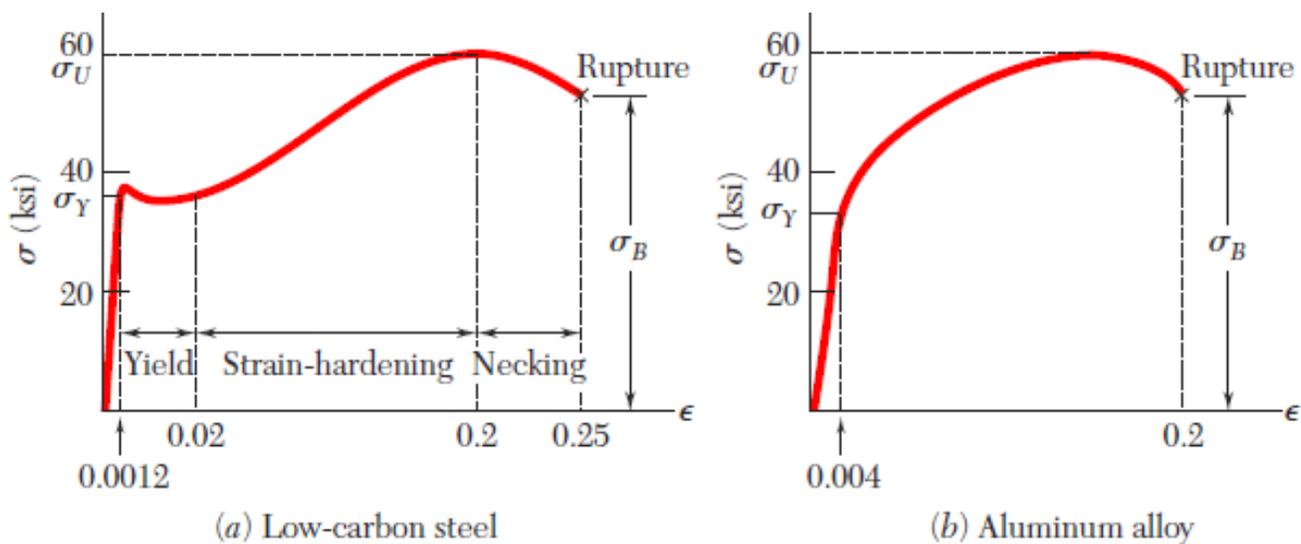


Figure (6.2): Stress-strain diagrams of two typical ductile materials

- ▶ The initial portion of the stress – strain diagram for most material used in engineering structures is a straight line. For the initial portion of the diagram, the stress (σ) is directly proportional to the strain (ϵ). Therefore, for a specimen subjected to a uniaxial load, can write:

$$\frac{\text{Stress}}{\text{Strain}} = E \rightarrow \sigma = E\epsilon$$

This relationship is known as *Hook's Law*.

- ❖ **Note:** *Hook's Law* describes only the initial linear portion of the stress – strain curve for a bar subjected to uniaxial extension.
- ❖ The slope of the straight line portion of the stress – strain diagram is called the *Modulus of Elasticity* or *Young's Modulus*.

$$E = \frac{\sigma}{\epsilon}$$

Deformation of a Body due to Force acting on it

Consider a body subjected to a tensile force as shown in Figure (6.3).

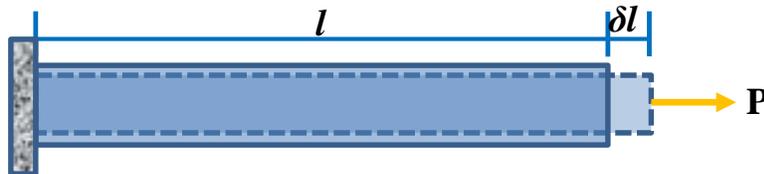


Figure (6.3)

Let P =Load or force acting on the body,

l =Length of the body,

A =Cross – sectional area of the body,

σ =Stress induced in the body,

E =Modulus of elasticity for the material of the body,

ϵ =Strain, and

δl =Deformation of the body.

Knowing that the stress is:

$$\sigma = \frac{P}{A}$$

And the strain is:

$$\varepsilon = \frac{\sigma}{E} = \frac{\frac{P}{A}}{E} = \frac{P}{EA}$$

Then the deformation is:

$$\delta l = \varepsilon(l) = \frac{Pl}{EA}$$

Examples

Example (6.1): A steel rod (1 m) long and (20 mmx20mm) in cross – section is subjected to a tensile force of (40 kN). Determine the elongation of the rod, if modulus of elasticity for the rod material is (200 GPa).

Solution:

$$\delta l = \frac{Pl}{EA} = \frac{40 \times 10^3 (1 \times 10^3)}{200 \times 10^3 (20 \times 20)} = 0.5 \text{ mm}$$

Example (6.2): In an experiment, a steel specimen of (13mm) diameter was found to elongate (0.2 mm) in a (200 mm) gauge length when it was subjected to a tensile force of (26.8 kN). If the specimen was tested within the elastic range, what is the value of Young’s modulus for the steel specimen?

Solution:

$$\delta l = \frac{Pl}{EA} \rightarrow 0.2 = \frac{26.8 \times 10^3 (200)}{E \left(\frac{\pi}{4} (13)^2 \right)}$$

$$E = 201910.2 \text{ MPa} \text{ or } 201.91 \text{ GPa}$$

Example (6.3): Determine the deformation of the steel rod shown in Figure (6.4) under the given loads. $E=29 \times 10^6$ psi.

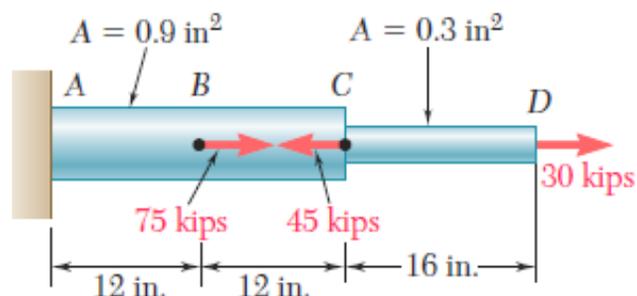


Figure (6.4)

Solution:

- We divide the rod into three component parts shown in Figure (6.5b) and write:

$$L_1 = L_2 = 12 \text{ in.}$$

$$L_3 = 16 \text{ in.}$$

$$A_1 = A_2 = 0.9 \text{ in}^2$$

$$A_3 = 0.3 \text{ in}^2$$

Section in each part as shown in Figure (6.5c), then:

$$\sum F_x = 0$$

$$P_1 = 60 \text{ kips} = 60 \times 10^3 \text{ lb}$$

$$P_2 = -15 \text{ kips} = -15 \times 10^3 \text{ lb}$$

$$P_3 = 30 \text{ kips} = 30 \times 10^3 \text{ lb}$$

The total deformation in steel bar is:

$$\delta L = \sum \frac{P_i L_i}{E_i A_i}$$

$$\delta L = \frac{1}{29 \times 10^6} \left(\frac{60 \times 10^3 (12)}{0.9} + \frac{-15 \times 10^3 (12)}{0.9} + \frac{30 \times 10^3 (16)}{0.3} \right) = \frac{2.2 \times 10^6}{29 \times 10^6} = 75.9 \times 10^{-3} \text{ in.}$$

Example (6.4): The rigid bar *BDE* is supported by two links *AB* and *CD*. Link *AB* is made of aluminum ($E=70 \text{ GPa}$) and has a cross-sectional area of 500 mm^2 ; link *CD* is made of steel ($E=200 \text{ GPa}$) and has a cross-sectional area of 600 mm^2 . For the 30-kN force shown in Figure (6.6), determine the deflection

- (a) of *B*,
- (b) of *D*,
- (c) of *E*.

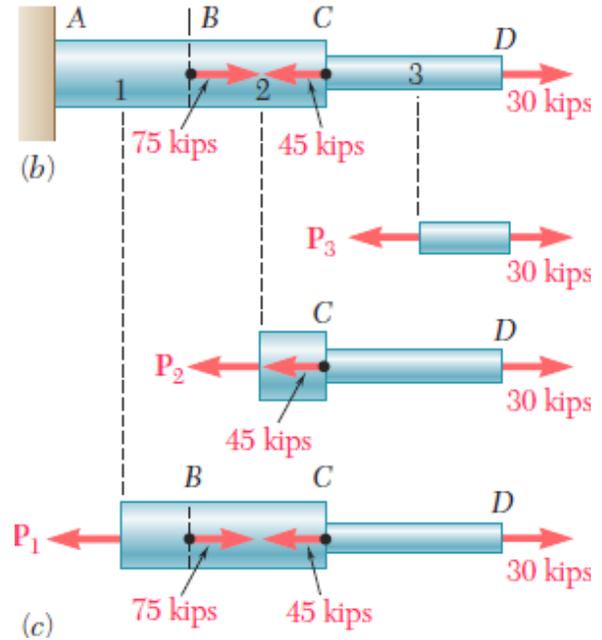


Figure (5.5)

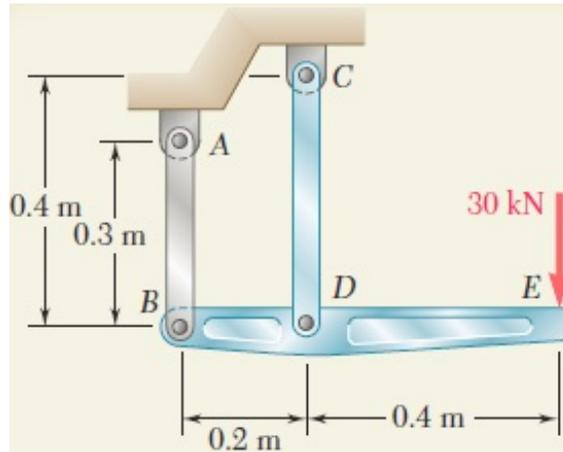


Figure (6.6)

Solution:

Bar BDE as F.B.D:

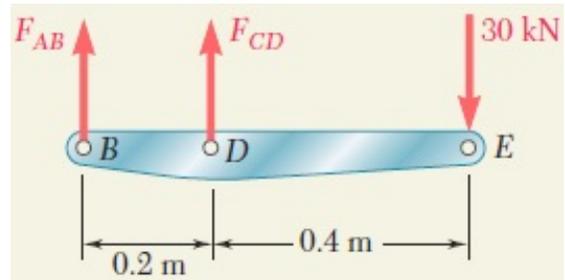
$$\circlearrowleft \sum M_B = 0$$

$$F_{CD}(0.2) - 30(0.6) = 0$$

$$F_{CD} = +90 \text{ kN (Tens.)}$$

$$\uparrow \sum F_y = 0 \rightarrow F_{AB} + F_{CD} - 30 = 0$$

$$\rightarrow F_{AB} = 30 - 90 = -60 \text{ (Comp.)}$$



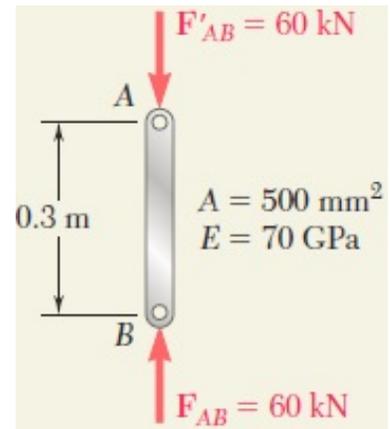
(a) Deflection of B:

$$\delta_B = \frac{Pl}{EA} = \frac{(-60 \times 10^3)(0.3)}{(70 \times 10^9)(500 \times 10^{-6})}$$

$$\delta_B = -514 \times 10^{-6} \text{ m or } \delta_B = -0.514 \text{ mm}$$

The negative sign indicates a contraction of member AB, and, thus, an upward deflection of end B:

$$\delta_B = 0.514 \text{ mm } \uparrow$$

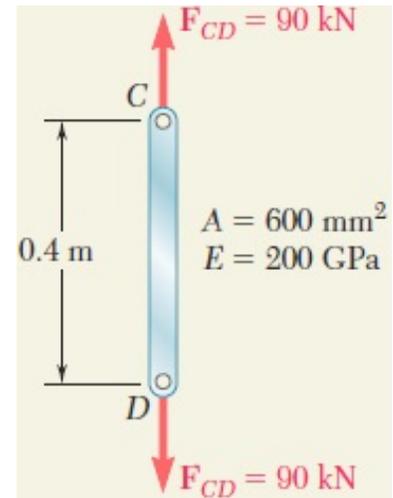


(b) Deflection of D:

$$\delta_D = \frac{Pl}{EA} = \frac{(90 \times 10^3)(0.4)}{(200 \times 10^9)(600 \times 10^{-6})}$$

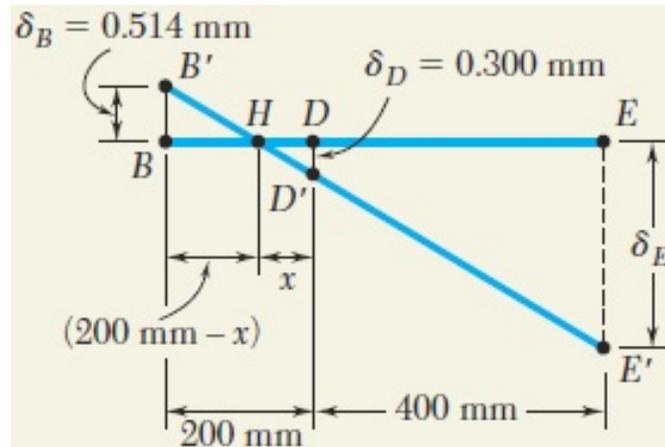
$$\delta_D = 300 \times 10^{-6} \text{ m or}$$

$$\delta_D = 0.300 \text{ mm} \downarrow$$



(c) Deflection of E:

We denote by B' and D' the displaced positions of points B and D . Since the bar BDE is rigid, points B' , D' , and E' lie in a straight line and we write:



$$\frac{BB'}{DD'} = \frac{BH}{HD} \rightarrow \frac{0.514}{0.300} = \frac{200 - x}{x} \rightarrow x = 73.7 \text{ mm}$$

$$\frac{EE'}{DD'} = \frac{HE}{HD} \rightarrow$$

$$\frac{\delta_E}{73.7} = \frac{400 + 73.7}{73.7}$$

$$\rightarrow \delta_E = 1.928 \text{ mm} \downarrow$$

Handwritten signature and name: Murtadha Al-Masoudy