

## EXAMPLES ON LONGITUDINAL AND TRANSVERSE VIBRATIONS

### امثلة تطبيقية على الاهتزازات الحرة الطولية والعرضية

#### Example .1.

A cantilever shaft 50 mm diameter and 300 mm long has a disc of mass 100 kg at its free end. The Young's modulus for the shaft material is 200 GN/m<sup>2</sup>. Determine the frequency of longitudinal and transverse vibrations of the shaft.

**Solution.** Given :  $d = 50 \text{ mm} = 0.05 \text{ m}$  ;  $l = 300 \text{ mm} = 0.3 \text{ m}$  ;  $m = 100 \text{ kg}$  ;  
 $E = 200 \text{ GN/m}^2 = 200 \times 10^9 \text{ N/m}^2$

We know that cross-sectional area of the shaft,

$$A = \frac{\pi}{4} \times d^2 = \frac{\pi}{4} (0.05)^2 = 1.96 \times 10^{-3} \text{ m}^2$$

and moment of inertia of the shaft,

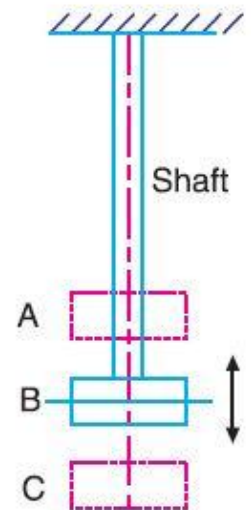
$$I = \frac{\pi}{64} \times d^4 = \frac{\pi}{64} (0.05)^4 = 0.3 \times 10^{-6} \text{ m}^4$$

#### Frequency of longitudinal vibration

We know that static deflection of the shaft,

$$\delta = \frac{Wl}{AE} = \frac{100 \times 9.81 \times 0.3}{1.96 \times 10^{-3} \times 200 \times 10^9} = 0.751 \times 10^{-6} \text{ m}$$

...( $\because W = m.g$ )



∴ Frequency of longitudinal vibration,

$$f_n = \frac{0.4985}{\sqrt{\delta}} = \frac{0.4985}{\sqrt{0.751 \times 10^{-6}}} = 575 \text{ Hz Ans.}$$

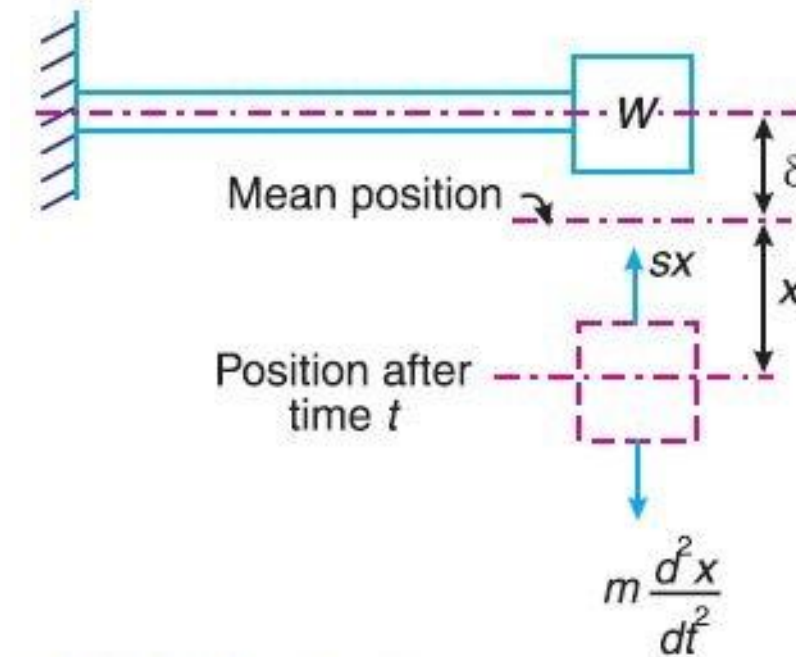
### Frequency of transverse vibration

We know that static deflection of the shaft,

$$\delta = \frac{W.l^3}{3EI} = \frac{100 \times 9.81 \times (0.3)^3}{3 \times 200 \times 10^9 \times 0.3 \times 10^{-6}} = 0.147 \times 10^{-3} \text{ m}$$

∴ Frequency of transverse vibration,

$$f_n = \frac{0.4985}{\sqrt{\delta}} = \frac{0.4985}{\sqrt{0.147 \times 10^{-3}}} = 41 \text{ Hz Ans.}$$



## Example .2.

A shaft of length 0.75 m, supported freely at the ends, is carrying a body of mass 90 kg at 0.25 m from one end. Find the natural frequency of transverse vibration. Assume  $E = 200 \text{ GN/m}^2$  and shaft diameter = 50 mm.

**Solution.** Given :  $l = 0.75 \text{ m}$  ;  $m = 90 \text{ kg}$  ;  $a = AC = 0.25 \text{ m}$  ;  $E = 200 \text{ GN/m}^2 = 200 \times 10^9 \text{ N/m}^2$  ;  $d = 50 \text{ mm} = 0.05 \text{ m}$

The shaft is shown in Fig. 2

We know that moment of inertia of the shaft,

$$I = \frac{\pi}{64} \times d^4 = \frac{\pi}{64} (0.05)^4 \text{ m}^4$$

$$= 0.307 \times 10^{-6} \text{ m}^4$$

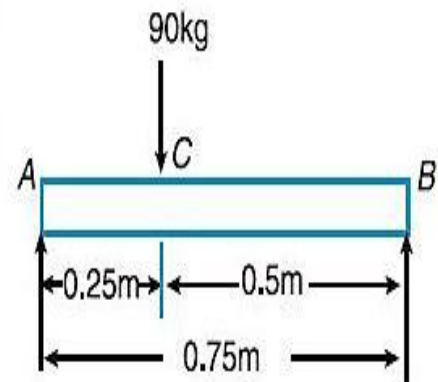


Fig. 2

and static deflection at the load point (i.e. at point C),

$$\delta = \frac{W a^2 b^2}{3EI l} = \frac{90 \times 9.81 (0.25)^2 (0.5)^2}{3 \times 200 \times 10^9 \times 0.307 \times 10^{-6} \times 0.75} = 0.1 \times 10^{-3} \text{ m}$$

... ( $\because b = BC = 0.5 \text{ m}$ )

### Example.3.

A flywheel is mounted on a vertical shaft as shown in Fig.3. The both ends of the shaft are fixed and its diameter is 50 mm. The flywheel has a mass of 500 kg. Find the natural frequencies of longitudinal and transverse vibrations. Take  $E = 200 \text{ GN/m}^2$ .

**Solution.** Given :  $d = 50 \text{ mm} = 0.05 \text{ m}$  ;  $m = 500 \text{ kg}$  ;  $E = 200 \text{ GN/m}^2 = 200 \times 10^9 \text{ N/m}^2$   
We know that cross-sectional area of shaft,

$$A = \frac{\pi}{4} \times d^2 = \frac{\pi}{4} (0.05)^2 = 1.96 \times 10^{-3} \text{ m}^2$$

and moment of inertia of shaft,

$$I = \frac{\pi}{64} \times d^4 = \frac{\pi}{64} (0.05)^4 = 0.307 \times 10^{-6} \text{ m}^4$$

#### Natural frequency of longitudinal vibration

Let  $m_1 =$  Mass of flywheel carried by the length  $l_1$ .

$\therefore m - m_1 =$  Mass of flywheel carried by length  $l_2$ .

We know that extension of length  $l_1$

$$= \frac{W_1 \cdot l_1}{A \cdot E} = \frac{m_1 \cdot g \cdot l_1}{A \cdot E} \quad \dots (i)$$

Similarly, compression of length  $l_2$

$$= \frac{(W - W_1) l_2}{A \cdot E} = \frac{(m - m_1) g \cdot l_2}{A \cdot E} \quad \dots (ii)$$

Since extension of length  $l_1$  must be equal to compression of length  $l_2$ , therefore equating equations (i) and (ii),

$$m_1 \cdot l_1 = (m - m_1) l_2$$

$$m_1 \times 0.9 = (500 - m_1) 0.6 = 300 - 0.6 m_1 \text{ or } m_1 = 200 \text{ kg}$$

$\therefore$  Extension of length  $l_1$ ,

$$\delta = \frac{m_1 \cdot g \cdot l_1}{A \cdot E} = \frac{200 \times 9.81 \times 0.9}{1.96 \times 10^{-3} \times 200 \times 10^9} = 4.5 \times 10^{-6} \text{ m}$$

We know that natural frequency of longitudinal vibration,

$$f_n = \frac{0.4985}{\sqrt{\delta}} = \frac{0.4985}{\sqrt{4.5 \times 10^{-6}}} = 235 \text{ Hz} \quad \text{Ans.}$$

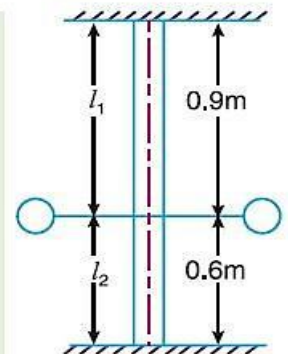


Fig. 3





### *Natural frequency of transverse vibration*

We know that the static deflection for a shaft fixed at both ends and carrying a point load is given by

$$\delta = \frac{Wa^3b^3}{3EI^3} = \frac{500 \times 9.81(0.9)^3(0.6)^3}{3 \times 200 \times 10^9 \times 0.307 \times 10^{-6}(1.5)^3} = 1.24 \times 10^{-3} \text{ m}$$

... (Substituting  $W = m.g$  ;  $a = l_1$ , and  $b = l_2$ )

We know that natural frequency of transverse vibration,

$$f_n = \frac{0.4985}{\sqrt{\delta}} = \frac{0.4985}{\sqrt{1.24 \times 10^{-3}}} = 14.24 \text{ Hz} \quad \text{Ans.}$$