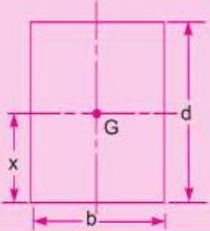
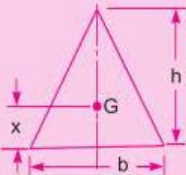


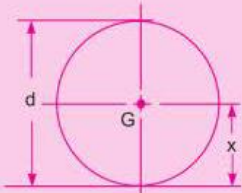
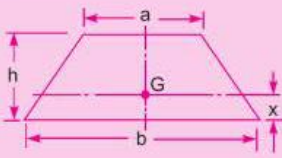
$P = \rho g h$ (where h is depth of the surface)

Table (3.1) The moments of inertia and other geometric properties of some important plane surfaces .

Plane surface	C.G. from the base	Area	Moment of inertia about an axis passing through C.G. and parallel to base (I_G)	Moment of inertia about base (I_0)
1. Rectangle 	$x = \frac{d}{2}$	bd	$\frac{bd^3}{12}$	$\frac{bd^3}{3}$
2. Triangle 	$x = \frac{h}{3}$	$\frac{bh}{2}$	$\frac{bh^3}{36}$	$\frac{bh^3}{12}$

Contd...

72 Fluid Mechanics

Plane surface	C.G. from the base	Area	Moment of inertia about an axis passing through C.G. and parallel to base (I_G)	Moment of inertia about base (I_0)
3. Circle 	$x = \frac{d}{2}$	$\frac{\pi d^2}{4}$	$\frac{\pi d^4}{64}$	—
4. Trapezium 	$x = \left(\frac{2a + b}{a + b} \right) \frac{h}{3}$	$\frac{(a + b)}{2} \times h$	$\left(\frac{a^2 + 4ab + b^2}{36(a + b)} \right) \times h^3$	—

3.3 / Inclined Plane surface submerged in liquid :

Consider a plane surface of arbitrary shape immersed in a liquid in such a way that the plane of the surface makes an angle Θ with the free surface of the liquid as shown in Fig.(3.2).

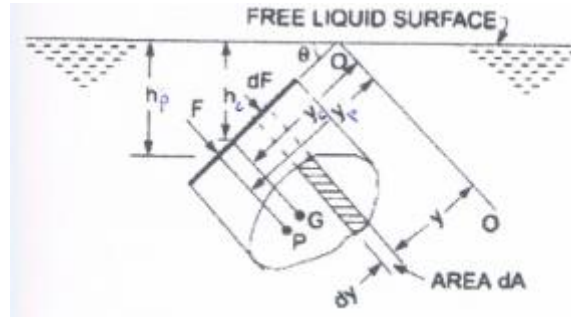


Fig.(3.2) Inclined immersed surface

Let , A total area of inclined surface , h_c depth of C.G of inclined area from free surface , h_p distance of center of pressure from free surface of liquid , Θ angle made by the plane of the surface with free surface , y_c distance of the C.G of the inclined surface from $O - O$, y_p distance of the center of pressure from the $O - O$.

Consider a small strip of area dA at a depth (h) from free surface and at a distance y from the axis $O - O$ as shown in Fig.(3.2).

$$\text{Force } dF \text{ on the strip} = p \times \text{Area of strip} = \rho g h \times dA$$

$$\text{Total Force on the whole area , } F = \int dF = \int \rho g h dA$$

$$\text{But from Fig.(3.2) , } \sin\Theta = \frac{h}{y} = \frac{h_c}{y_c} = \frac{h_p}{y_p}$$

$$\text{Therefore , } h = y \sin \Theta$$

$$F = \int \rho g \times y \sin\Theta \times dA = \rho g \sin \Theta \int y dA$$

$$\text{But , } \int y dA = A y_c$$

$$\text{Therefore , } F = \rho g \sin \Theta \times A \times y_c$$

$$F = \rho g A h_c \quad (3.6)$$

$$\text{Force on the strip , } dF = \rho g h dA$$

$$\sin \Theta = \frac{h}{y}, \quad h = y \sin \Theta$$

$$dF = \rho g y \sin\theta dA$$

Moment of force (dF) about axis O – O ,

$$dF \times y = \rho g y \sin\theta dA \times y = \rho g \sin\theta y^2 dA$$

Sum of moments of all such forces about O - O ,

$$M = \int \rho g \sin\theta y^2 dA = \rho g \sin\theta \int y^2 dA$$

$$\text{But } \int y^2 dA = I_o$$

$$\text{Therefore , } M = \rho g \sin\theta I_o \quad (3.7)$$

Moment of the total force F , about O – O is given by : $F \times y_p$ (3.8)

Equating the two values given by equations (3.7) & (3.8)

$$F \times y_p = \rho g \sin\theta I_o$$

$$y_p = \frac{\rho g \sin\theta I_o}{F} \quad (3.9)$$

$$\text{But , } \sin\theta = \frac{h_p}{y_p} \quad , \quad y_p = \frac{h_p}{\sin\theta} \quad , \quad \text{and } F = \rho g A h_c$$

And $I_o = I_G + A y_c^2$, Substituting these values in eq.(3.9) , we get :

$$\frac{h_p}{\sin\theta} = \frac{\rho g \sin\theta}{\rho g A h_c} (I_G + A y_c^2) \quad (\times \sin\theta)$$

$$\text{But , } \sin\theta = \frac{h_c}{y_c} \quad , \quad y_c = \frac{h_c}{\sin\theta}$$

$$h_p = \frac{\sin^2\theta}{A h_c} (I_G + A \frac{h_c^2}{\sin^2\theta})$$

$$h_p = \frac{I_G \sin^2\theta}{A h_c} + h_c \quad (3.10)$$

If the $\theta = 90^\circ$, equation (3.10) becomes same as equation (3.5) (vertical plane submerged) .

3.4 / Curved Surface Submerged in Liquid :

Consider a curved surface (AB) , submerged in a static liquid as shown in Fig.(3.3) . Let dA is the area of a small strip at a depth of (h) from water surface.

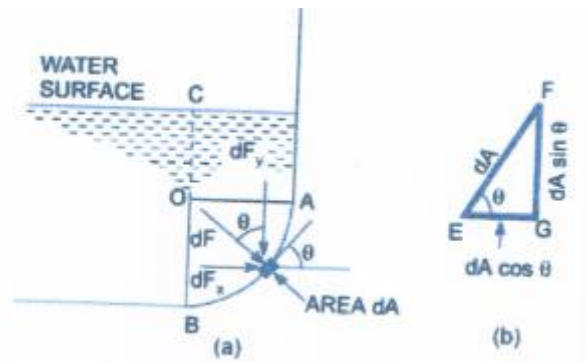


Fig.(3.3)

Then pressure (p) = $\rho g h$

Force (dF) = $p \times \text{area} = \rho g h \times dA$ (3.11) This force dF acts normal to the surface , hence , total force on the curved surface should be:

$$F = \int \rho g h dA \quad (3.12)$$

By resolving the force dF in two components dF_x and dF_y in the x and y directions respectively . The total force in the x and y directions , i.e , F_x and F_y are obtained by integrating dF_x and dF_y , Then total force on the curved surface is :

$$F = \sqrt{F_x^2 + F_y^2} \quad (3.13)$$

And inclination of resultant with horizontal is ,

$$\tan \Theta = \frac{F_y}{F_x} \quad (3.14)$$

Resolving the force dF given by equation (3.11) in x and y directions :

$$dF_x = dF \sin\Theta = \rho g h dA \sin\Theta$$

$$dF_y = dF \cos \Theta = \rho g h dA \cos \Theta$$

Total forces in the x and y directions are :

$$F_x = \int dF_x = \rho g \int h dA \sin\Theta \quad (3.15)$$

$$F_y = \int dF_y = \rho g \int h dA \cos \Theta \quad (3.16)$$

Fig.(3.3) b , shows the enlarged area dA , from this figure , i.e. , ΔEFG :

$$EF = dA \quad , \quad FG = dA \sin\Theta \quad , \quad EG = dA \cos\Theta$$

Thus , in Eq.(3.15) , $dA \sin\Theta = FG =$ Vertical projection of the area dA .

Therefore , F_x force on the projected area on the vertical plane .

Thus , in Eq.(3.16) , $dA \cos\Theta = EG =$ Horizontal projection of the area dA .

Therefore , $\int h dA \cos\Theta$ is the total volume contained between the curved surface , extended up to free surface .

Hence , $\rho g \int h dA \cos\Theta$ is the total weight supported by the curved surface , thus
 , $F_y = \rho g \int h dA \cos \Theta =$ Weight of liquid supported by the curved surface up to free surface of liquid.