The general procedure for use of Table 11-2 is as follows:

1. Determine the air-flow requirements and the room size.

- 2. Select the number, location, and type of diffuser to be used.
- 3. Determine the room characteristic length.
- 4. Select the recommended throw-to-length ratio from Table 11-2.
- **5.** Calculate the throw.
- 6. Select the appropriate diffuser from catalog data such as those in Tables 11-3, 11-4, 11-5, or 11-6.
- 7. Make sure any other specifications are met (noise, total pressure, etc.).

Ex: The room shown in Figure (1) is part of a single-story office building located in the central United States. A perimeter air-distribution system is used. The air quantity required for the room is **250 cfm**. Select diffusers for the room based on cooling.

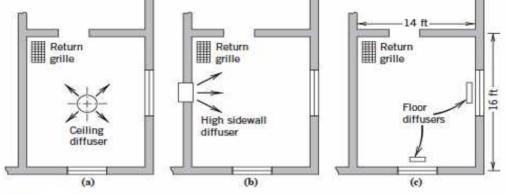


Figure 1 Plan view of a room showing location of different types of outlets.

Solution:.

Diffusers of the type shown in **Table 11-3** should be used for this application.

A diffuser should be placed under each window in the floor near the wall (Figure *c*) because the room has two exposed walls. This will promote mixing with the warm air entering through the window. The total air quantity is **divided equally** between the two diffusers (250/2=125 cfm). If we assume that the room has an 8 ft ceiling and a room cooling load of 40 Btu/(hr-ft2), the room characteristic length is 8 ft (16/2).

Table 11-2 gives a throw-to-length ratio of 1.3 for a straight vane diffuser. Then

$$\frac{x50}{L} = 1.3 \rightarrow \frac{x50}{8} = 1.3 \rightarrow \underline{x50} = 10.4 \text{ ft} \text{ (throw=blow)}$$

From Table 11-3, <u>a 4 × 12 in.</u> diffuser with <u>125 cfm</u> has a throw (blow), corrected for length, between

$$x50 = 13(\frac{3}{4}) = 9.7$$
 ft and $x50 = 17(\frac{3}{4}) = 12.7$ ft

because 125 cfm lies between 111 cfm and 139 cfm.

The NC is quite acceptable and is between $\underline{12 \text{ and } 18}$, uncorrected for length. The total pressure required by the diffuser is between 0.036 and 0.057 in. wg and is about

$$\Delta P = (125/111)^2 \times (0.036) = 0.046$$
 in. wg

An acceptable solution is listed as follows:

Size, in.	Capacity, cfm	Throw, ft	NC	ΔP_0 , in. wg	
4×12	125	10.5	<15	0.046	

The loss in total pressure for the diffuser is an important consideration. The value shown above would be acceptable for a light commercial system.

Diffuser Type	Characteristic Length L					
High sidewall grille	Distance to wall perpendicular to jet					
Circular ceiling diffuser	Distance to closet wall or intersecting air jet					
Sill grille	Length of room in direction of jet flow					
Ceiling slot diffuser	Distance to wall or midplane between outlets					
Light troffer diffusers	Distance to midplane between outlets plus distance from ceiling to top of occupied zone					
Perforated, louvered ceiling diffusers	Distance to wall or midplane between outlets					

Table 11-1 Characteristic Room Length for Several Diffusers

Source: Reprinted by permission from ASHRAE Handbook, Fundamentals Volume, 1997.

	Room	x_{50}/L^a for		For ADPI	
Terminal Device	Load, Btu/hr-ft ²	Maximum ADPI	Maximum ADPI	Greater Than	Range of x ₅₀ /L ^a
High sidewall	80 (252)	1.8	68		199 <u>9</u>
grilles	60 (189)	1.8	72	70	1.5-2.2
	40 (126)	1.6	78	70	1.2-2.3
	20 (63)	1.5	85	80	1.0-1.9
Circular ceiling	80 (252)	0.8	76	70	0.7-1.3
diffusers	60 (189)	0.8	83	80	0.7-1.2
	40 (126)	0.8	88	80	0.5-1.5
	20 (63)	0.8	93	90	0.7-1.3
Sill grille,	80 (252)	1.7	61	60	1.5-1.7
Straight vanes	60 (189)	1.7	72	70	1.4-1.7
1 Contraction of the second	40 (126)	1.3	86	80	1.2-1.8
	20 (63)	0.9	95	90	0.8-1.3
Sill grille,	80 (252)	0.7	94	90	0.6-1.5
Spread vanes	60 (189)	0.7	94	80	0.6-1.7
	40 (126)	0.7	94	<u> </u>	
	20 (63)	0.7	94		
Ceiling slot	80 (252)	0.3	85	80	0.3-0.7
diffusers	60 (189)	0.3	88	80	0.3-0.8
$(for T_{100}/L)^{a}$	40 (126)	0.3	91	80	0.3-1.1
100	20 (63)	0.3	92	80	0.3-1.5
Light troffer	60 (189)	2.5	86	80	<3.8
diffusers	40 (126)	1.0	92	90	<3.0
	20 (63)	1.0	95	90	<4.5
Perforated and	11-51 (35-160)	2.0	96	90	1.4-2.7
louvered ceiling diffusers	2012 80400 MC24 5405 864 1		Displat A	80	1.0-3.4

Table 11-2 Air Diffusion Performance Index (ADPI) Selection Guide

"For SI units, $x_{0.25}/L$ and $T_{0.5}/L$

Source: Reprinted by permission from ASHRAE Handbook, Fundamentals Volume, 1997.

Size,	A	Total	Flow.			Throw, ^a ft	
in.	Area, ft ² /ft	Pressure, in. wg	cfm/ft	NC ^b	Min.	Mid.	Max.
2	0.055	0.009	22		1	1	1
		0.020	33	·		4	4
		0.036	44	12	4	7	7
		0.057	55	18	9	9	10
		0.080	66	23	11	11	12
		0.109	77	27	13	14	16
		0.143	88	31	14	16	18
		0.182	99	34	15	17	20
		0.225	110	37	17	19	21
4	0.139	0.009	56	_	3	3	3
		0.020	83		9	9	9
		0.036	111	12	13	13	13
		0.057	139	18	16	16	17
		0.080	167	23	20	20	21
		0.109	195	27	22	23	24
		0.143	222	31	24	25	26
		0.182	250	34	27	27	27
		0.225	278	37	30	30	30
6	0.221	0.009	88		5	5	5
		0.020	133	·	10	10	10
		0.036	177	13	15	15	15
		0.057	221	19	18	18	18
		0.080	265	24	23	23	23
		0.109	310	28	25	25	25
		0.143	354	32	28	28	28
		0.182	398	35	31	31	31
		0.225	442	38	32	32	32

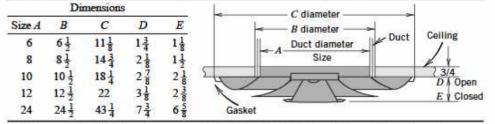
Table 11-3 Performance Data for a Typical Linear Diffuser

Active Length,		blier Factor for Th Terminal Velocity		
ft	150	100	50	נייייון אורי ביורי ביורי ואוניייי
1 10 or continuous	0.5 s 1.6	0.6 1.4	0.7 1.2	
Active Length, ft	NC Correction	Active Length,	NC Correction	Size
1	-10	10	0	
2	-7	15	+2	
4	-4	20	+3	
6	-2	25	+4	
8	-1	30	+5	

^aMinimum throw values refer to a terminal velocity of 150 fl/min, middle to 100 fl/min, and maximum to 50 fl/min, for a 4 ft active section with a cooling temperature differential of 20 F. The multiplier factors listed at the bottom are applicable for other lengths. ^bBased on a room absorption of 80 dB referred to 10⁻¹² W, and a 10 ft active section. *Source*: Reprinted by permission of Environmental Elements Corporation, Dallas, TX.

Size,	Neck Velocity,	Velocity Pressure,	Total Pressure,	Flow Rate,	Radiu	s of Diffu	sion,ª ft	
in.	ft/min	in.wg	in. wg	cfm	Min	Mid	Max.	NC ⁶
6	400	0.010	0.026	80	2	2	4	
	500	0.016	0.041	100	2	3	5	_
	600	0.023	0.059	120	2	4	6	14
	700	0.031	0.079	140	3	4	7	19
	800	0.040	0.102	160	2 2 3 3 4	5	8	23
	900	0.051	0.130	180	4	5	9	26
	1000	0.063	0.161	200	4	6	10	30
	1200	0.090	0.230	235	5	7	11	35
8	400	0.010	0.033	140	2	4	6	
ಂಕಾಂ	500	0.016	0.052	175	5 2 3	4	7	15
	600	0.023	0.075	210	4	5	9	21
	700	0.031	0.101	245	4	6	10	26
	800	0.040	0.130	280	5	7	n	31
	900	0.051	0.166	315	5 5	8	13	34
	1000	0.063	0.205	350	6	9	14	37
	1200	0.090	0.292	420	7	11	17	44
10	400	0.010	0.027	220	3	4	7	
07.1	500	0.016	0.043	270	3	5	8	11
	600	0.023	0.062	330	7 3 4 5	6	10	17
	700	0.031	0.084	380	5	7	11	21
	800	0.040	0.108	435	5	8	13	26
	900	0.051	0.138	490	6	9	15	30
	1000	0.063	0.170	545	7	10	16	33
	1200	0.090	0.243	655	8	12	20	39
12	400	0.010	0.026	315	3	5	8	
-1276-0	500	0.016	0.042	390	4	6	10	11
	600	0.023	0.060	470	5	7	12	17
	700	0.031	0.081	550	5	8	13	22
	800	0.040	0.105	630	6	10	15	26
	900	0.051	0.134	705	7	11	17	30
	1000	0.063	0.166	785	8	12	19	33
	1200	0.090	0.236	940	10	14	23	39
18	400	0.010	0.030	710	5	7	12	
177712	500	0.016	0.048	885	6	9	15	15
	600	0.023	0.069	1060	7	11	18	21
	700	0.031	0.093	1240	9	13	21	26
	800	0.040	0.120	1420	10	15	24	30
	900	0.051	0.153	1590	11	17	27	34
	1000	0.063	0.189	1770	12	19	30	37
	1200	0.090	0.270	2120	15	22	36	43
0.4	100	0.010	0.004	10.00			10	

Table 11-4 Performance Data for a Typical Round Ceiling Diffuser (continued)



"Minimum radii of diffusion are to a terminal velocity of 150 ft/min, middle to 100 ft/min, and maximum to 50 ft/min. ^bThe NC values are based on a room absorption of 18 dB referred to 10⁻¹³ W (8 dB referred to

10-12 W).

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Sizes.	A.	Flow, Rate,	Veloc.	Veloc. Press.	То	tal Press in wg	ire,		Defi.	-	Throw,	fi
in.	ft ²	cfm	ft/min	in.wg	0°	22 ± °	45°	NC	deg	Min	Mid.	Max
8×4	0.18	70	400	0.010	0.017	0.019	0.029		0	6	8	15
7×5,									224	5 3 7	6	12
6×6									45	3	4	8
10×4	0.22	90						_	0	7	10	17
8×5,									224	6 3	8	14
7×6									45	3	5	9
12×4 ,	0.26	105						_	0	7	11	19
10×5,	2,548.1								$22\frac{1}{2}$	6	9	15
8×6									45	4	5	9
16×4,	0.34	135							0	8	12	21
12×5,	10000100	0.000							22+	6	10	17
10×6									45	4	6	11
18×4,	0.39	155							40 0 22 45	4 9 7	13	23
14×5,								2000	224		10	18
12×6.			110000	0.000	. Japan a	and the second			45	4	6	11
8×4,	0.18	90	500	0.016	0.028	0.031	0.047		0	1	11	17
7×5.									22]	6	9	14
6×6									45	4	5	9
10×4.	0.22	110							0	8	12	19
8×5,									22	6	10	15
7×6									45	4	6	10
12×4	0.26	130							0	9	13	21
10×5.									224	7	10	17
8×6									45	4	7	10
16×4.	0.34	170						$\frac{1}{2} = \frac{1}{2} = \frac{1}{2}$	0	10	15	24
12×5.									$22\frac{1}{2}$	8	12	19
10×6									45	5	8	11
18×4,	0.39	195						(((((((((((((((((((0	11	16	25
14×5.	883	82255							224		13	20
12×6,									45	9 5	8	13

Table 11-5 Performance Data for an Adjustable-Type, High Sidewall Diffuser

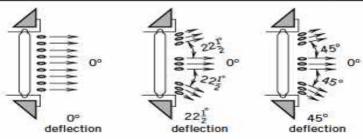
continues

Table 11-5 Performance Da	ata for an Adjustable-Typ	e, High Sidewall Diffuser	(continued)

Sizes.	40	Flow, Rate.	Veloc.,	Veloc. Total Pressure, Press., in. wg				Defl.	Throw, ft			
in.	ff ²	cfm	ft/min	in.wg	0°	22 1°	45°	NC	deg	Min.	Mid	Max
8×4,	0.18	110	600	0.022	0.038	0.043	0.064	10	0	9	13	19
7×5,									22	7	10	15
6×6									45	4	7	10
10×4,	0.22	130						10	0	9	15	21
8×5,									22	7	12	17
7×6									45	5	7	10
12×4,	0.26	155						11	0	10	16	23
10×5,									22	8	13	18
8×6									45	5	8	11
16×4,	0.34	205						12	0	12	19	26
12×5.									22 +	10	15	21
10×6									45	6	9	13
18×4,	0.39	235						13	0	13	19	28
14×5.									224	10	15	22
12×6.									45	7	10	14
8×4.	0.18	125	700	0.030	0.052	0.058	0.088	15	0	10	15	20
7×5.									22 +	8	12	16
6×6									45	5	7	10
10×4,	0.22	155						15	0	11	16	23
8×5.									221	9	13	18
7×6									45	6	8	11
12×4,	0.26	180						16	0	12	17	24
10×5.	100.000	1.2388						1992	224	10	14	19
8×6									45	6	9	12
16×4,	0.34	240						17	0	14	20	28
12×5.	and the second	- 38						262.5	224	11	16	22
10×6									45	7	10	14
18×4,	0.39	275						18	0	15	22	30
14×5,	37777.0	200						1.222	22 +	12	18	24
12×6.									45	8	11	15
8×4.	0.18	145	800	0.040	0.069	0.078	0.117	19	0	11	16	22
7×5.	Contraction of the second seco	2,2420	25226	202,020	10000000	1000	1000000	0.570	22 1	9	13	18
6×6									45	6	8	11
10×4.	0.22	175						19	0	13	17	24
8×5.		-							221	10	14	19
7×6									45	6	9	12
12×4.	0.26	210						20	0	14	19	26
10×5.								~~~	224	11	15	21
8×6									45	7	9	13
16×4.	0.34	270						21	õ	16	22	30

Table 11-5 Performance Data for an Adjustable-Type, High Sidewall Diffuser (continued)

Sizes,	<i>A</i> -	Flow, Rate.	Veloc.,	Veloc. Press	Te	in. wg	ue,		Defl.	Throw, ft		ft
in.	AS AS	cfm	ft/min	in. wg	0°	22 ¹ / ₂ °	45°	NC	deg	Min.	Mid.	Max.
10×4	0.22	220				200		25	0	16	19	27
8×5,									22 1	13	15	22
7×6									45	8	10	13
12×4,	0.26	260						26	0	17	21	19
10×5,									22 1	14	17	23
8×6									45	8	11	15
16×4,	0.34	340						27	0	20	24	33
12×5,									22 1	16	19	26
10×6									45	10	12	17
18×4,	0.39	390						28	0	21	26	36
14×5,									22 1	17	21	29
12×6,									45	11	13	18



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<u>Example 2</u>: Suppose the room of Figure 1 is located in the southern latitudes where overhead systems are recommended. Select a round ceiling diffuser system and a high sidewall system. Also select a return grille.

Given: 250 cfm air quantity Required:

Select a round ceiling diffuser, select high sidewall grille, and select a return grille.

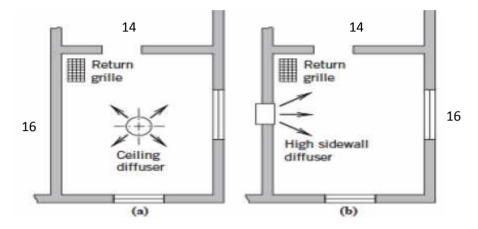


Figure 1

Solution: The data of Table 11.1 with information from Table 11.2 and 11.4 will be used to select a **ceiling diffuser**. The characteristic length is 7 or 8 ft and the throw-to-length ratio is 0.8; then

 $x_{50} / L = 0.8 \rightarrow x_{50} = 0.8 \times (7) = 5.6 \text{ ft}$

Using correction factor: $x_{50}=5.6/0.75=7.5$

The best choice would be

Size, in	Throw, ft	NC	△P ₀ , in. wg
10	7½	10	0.035

The throw is larger than desired, but the throw-to-length ratio is within the range to give a minimum *ADPI* of 76 percent. **Figure 1a** shows this application.

A high sidewall diffuser may be selected from Table 11.2. In this case the throw-to-length ratio should be about 1.8 and the characteristic length is 14 ft; then

 $x_{50} / L = 1.8 \rightarrow x_{50} = 1.8 \times (14) = 25.2 \text{ ft}$

At 240 cfm, pressure drop at 22 ½ degree spread would be 0.058: At 250 cfm, pressure drop at 22 ½ degree spread would be acceptable

$$\Delta P = (\frac{250}{240})^2 \times 0.058 = 0.063 \text{ in. wg}$$

The best choice would be

Size, in	Throw, ft	NC	⊿P₀, in. wg
16 x4			
12 x 5	25	18	0.063
10 x 6	_		

RETURN GRILLES

Velocities thru return grilles depend on (1) the static pressure loss allowed and (2) the effect on occupants or materials in the room. In determining the pressure loss, computations should be based on the free velocity thru the grille, not on the face velocity, since the orifice coefficient may approach 0.7. In general the following velocities may be used (see table 1-7):

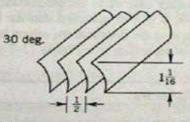
GRILLE LOCATION	FPM OVER GROSS AREA
Commercial	
Above occupied zone	800 and above
Within occupied zone not near seats	600-800
Within occupied zone near seats	400-600
Door or wall louvers	500-1000
Undercutting of doors	600 •
Industrial	800 and above
Residential	400

•Thru undercut area

		Core Velocity, fpm	200	300	400	500	600	700	808
		Velocity Pressure, in. wg		0.006	0.010	0.016	0.023	0.031	
		Static	-		1000			0.031	0.04
A 11 ²		Pressure,							
ft2	Sizes, in.	in. wg	-0.011	-0.033	-0.055	-0.088	-0.126	-0.170	-0.2
0.34	16×4	cfm	70	100	135	170	205	240	
	10×6	NC ^a			13	20	25	30	270
0.39	18×4	cfm	80	115	155	195	235	275	33
	12×6	NC			14	21	26	31	310
0.46	20×4	cfm	90	140	185	230	275	320	34
	14×6	NC			15	22	27		37(
	10×8	a contraction of the second						32	35
0.52	24×4	cfm	105	155	210	260	310	360	1.10
	16×6	NC	0.000		16	23	28	365	415
0.60	28×4	cfm	120	180	240	300		33	36
	18×6	NC		100	17	24	360	420	480
	12×8					24	29	34	37
0.69	30×4	cfm	140	205	275	240		AL SHOW	
	20×6	NC	140	205	275	345	415	485	550
	14×8				17	24	29	34	37
	12×10								1
0.81	36×4	afre.	160		- Carton	-			
	22×6	cfm	160	245	325	405	485	565	650
	16×8	NC		10	18	25	30	35	38
									50
0.90	14×10								
0.50	40×4	cfm	180	270	360	450	540	630	720
	26×6	NC		11	19	26	31	36	
	18×8							50	39
	16×10								
	12×12								
1.07	48×4	cfm	215	320	430	535	-	-	-
	30×6	NC		12	20		640	750	855
	18×10				20	27	32	37	40
	14 × 12								
1.18	34×6	cfm	235	255	170	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1			
	24×8	NC	200	355	470	590	710	825	945
	20×10			13	21	28	33	38	41
	16×12							Sen 19	
1.34	60×4	cfm	270						
-	36×6		270	400	535	670	805	940	1070
	18×12	NC		13	21	28	33	38	
	16×14				The later			.70	41
.60		Contraction of the local division of the loc							
	30×8	cfm	320	480	640	800	0.00		-
	24×10	NC		14	22	800	960	1120	1280
	22×12			an tea	**	29	34	39	42
	18×14								

	le 11 .6Perfor	Core Velocity, fpm	200	300	400	500			
		Velocity Pressure, in. wg	0.002	0.006	0.010	0.016	600	700	800
A _c ft ²	Sizes, in.	Static Pressure, in, wg	-0.011	0.011 -0.033 0.055 0.031		0.040			
1.80	48×6	cfm	360	540		-0.088	-0.126	-0.170	-0.220
	36×12 30×10 24×12	NC		15	720 23	900 30	1080 35	1260 40	1440 43
2.08	60×6 40×8 36×10 30×12	efm NC	415	625 16	830 24	1040 31	1250 36	1460 41	1660 44
	24×14 20×16								
2.45	48 × 8 26 × 14 24 × 16	cfm NC	490	735 17	980 25	1220 32	1470 37	1720 42	1960 45
2.78	36 × 12 30 × 14	cfm NC	555	835 18	1110 26	1390 33	1670 38	1950	2220
	26×16 24×18				-	1	30	43	46
3.11	40×12 36 × 14 30 × 16	cfm NC	620	935 19	1240 27	1560 34	1870 39	2180 44	2490 47
3.61	24 × 20								
.01	48 × 12 36 × 16 24 × 24	cfm NC	720	1080 20	1440 28	1800 35	2170 40	2530 45	2890 48

*Based on a room absorption of 8 dB, with respect to 10-12 watts, and one return.



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Example: Small store dimensions: 32 × 23 × 16 ft Ceiling – flat Load – equally distributed Air quantity – 2000 cfm Temp difference – 25 F **Find:** Number of outlets, Size of outlets,

Solution:

-The minimum blow is 75% of the room width for the given condition of equally distributed heat load. Therefore, the minimum blow necessary is: $23 \times 0.75=17.3$ ft

- The maximum blow is the width of the room =32 ft

- The blow of 17.5 to 34 ft.
- No. of outlets=
$$\frac{2000}{500} = 4$$

- nominal size 24 in. x 6 in
 $k = \frac{2000}{32 \times 16} = 3.9$

TABLE 1-8 - WALL OUTLET RATINGS, FOR COOLING ONLY

For Flat Ceilings

OUTLET VELOCITY 250 FPM STATIC PRESSURE STANDARD OUTLET Sir 8 = .01, 2214" = .01 STANDARD OUTLET 45" = .01 STATIC PRESSURE WITH METERING PLATE Sir 8 = .01, 2214" = .015		375 FFM				SOO FPM					730 PPM										
			¢1	See 8 = .013, 2236 * = .015 45* = .019					SH 8 = .024, 2255 ⁺ = .028 45 ⁺ = .025				Ser 8 = .051, 2215" = .061 45" = .08								
						5e 1024, 22 %043 45"065				Ser 8 = .061, 22 %" = .082 45" = .118				682	Ser 8 = .175, 22% * = .19 45* = .37						
Nam, Size of Outlet Vane (and Pres Setting Area)		Air Quan- Hity (sfm)	Blew (h)	15	p Diff 20 A Cla	25	Ab Quan- Illy (cfm)	81ew (%)	15	p Diff 20 a Cig	35	Air Quan- Iiiy (cfm)	84aw (11)	15	20 20 Cip	75	Alr Quen- Miry (ctm)	514w (P)	15	p Diñ 20 n Cip	25
8 x 4 (16.9)	Straight 2214* 45*	30	33 23 14	6.5 6.5 6.0	7.0 6.5 6.0	7.0 4.5 4.5	-	7.0 5.1 3.5	73	7.5 7.0 4.5	8.0 7.0 7.0	59	10.0 7.5 -5.0	75 70 63	4.0 7.5 6.5	11.5 7.5 7.9	10	17.0 13.0 9.0	15 75 65	90 7.5 7.0	9.0 6.0 7.0
19 x 4 (71.7)	Strolght 22 %* 45*	37	1.5 2.5 1.8	8.5 6.5 6.0	7.0 6.5 6.0	7.5 7.0 6.5	. 17 .	7.4 5.5 3.7	7.5 6.5 6.3	7.5 7.0 6.5	6.0 7.0 7.0	75	10.5 8.0 5.4	7.5 7.0 6.5	8.0 7.5 6.5	4.5 7.5 7.9	392	18.0 13.0 9.0	1.5 7.5 6.5	9.0 8.0 7.0	9.0 8.0 7.0
12 x 4 (24.6)	Stratght 221%* 45*	-	15 25	63 63 69	7.0	7.5		7.5	7.5 7.0 4.5	7.5	8.0 7.3 7.0	95	11.0 8.1 5.5	8.0 7.0 6.5	8.0 7.5 7.0	13 73 70	136	18.0 13.0 9.0	8.5 7.5 6.5	9.0 8.0 7.0	9.5 8.3 7.0
16 x 4 (33.9)	Straight 221/5* 45*	61	17 27 10	7.0 6.5 6.0	7.0 6.5 6.5	7.5	12	7.9	7.5 7.0 6.5	7.5	8.0 7.5 7.0	122	11.0 8,1 3.5	10 7.0 6.5	10 7.5 7.0	11.5 7.5 7.0	183	19,0 14,0 10,0	15 7.5 6.5	9.0 8.0 7.0	1.1 8.3 7.5
20 x 4 (45.5)	Straight 22%* 43*	77	4.0 3.0 2,0	7.0 4.5 6.0	7.0 6.5 6.5	7.3 7.0 6.3	310	8.0 6.0 4.0	7.5 7.0 6.5	4.0 7.0 4.5	8.0 7.5 7.0	364	11.5 8.5 .6.0	8.0 7.5 6.5	8.0 7.3 7.0	8.5 8.0 7.0	221	20.0 15.0 10.0	***	9.0 8.0 7.0	9.5 8.3 7.3
24 x 4 (35.0)	Straight 221/1" 45"	93	4.1 3.1 3.0	7.0 6.5 6.0	7.0 7.0 6.5	7.5 7.0 8.5	139	8.0 6.0 4.0	7.5 7.0 6.5	8.0 7.0 6.5	8.0 7.5 7.0	165	11.5 8.5 6.0	8.0 7.5 6.5	8.0 7.5 7.0	8.5 8.0 7.0	271	20.0 15.0 10.6	4.5 7.5 4.5	9,0 8,0 7,0	10.0 8.3 7.3
30 x 4 (68.3)	Strolgta 22%* 43*	116	4.2 3.1 3.1	7.0 6.5 6.0	7.0 7.0 6.5	7.5 7.0 4.5	175	8.0 6.0 4.0	7.5 7.0 6.5	8.0 7.5 6.5	8.0 7.5 7.0	233	12.0 9.0 6.0	8.0 7.5 6.5	8.0 7.5 7.0	1.5 1.0 7.0	549	21.0 16.0 11.0	8.3 7.5 7.0	9,5 8,0 7,0	10.0 8.3 7.3
36 x 4 (83,5)	Strolght 22 % * 45*	140	4.4 3.3 2.2	7.0 6.5 6.0	7.5 7.0 6.5	7.3 7,0 6,5	210	8.0 4.0 4.0	7.5 7.9 6.5	8.0 7.5 6.5	8.0 7.5 7.0	279	12.0 9.0 6.0	8.0 7.5 6.5	1.5 7.5 7.0	9,0 8,0 7,0	420	21.0 16.0 11.0	9.0 7.5 7.0	9.5 6.5 7.0	10.0 8.2 7.2
(24.3)	Straight 2216* 45*	33	5.0 3.8 2.5	7.5 7.0 6.0	7.5 7.0 6.5	8.0 7.5 6.5	n	\$.5 7.0 4.8	8.0 7.0 6.5	8.0 7.3 7.0	8.5 6.0 7.0	103	13.0 10.0 6.0	8.5 7.5 7.0	9.0 8.0 7.0	9.0 6.5 7.5	135	24.0 16.0 12.0	8.5 8,0 7,0	10.0 8.5 7.5	10.5 9.5 8.0
10 x 6 (34,0)	Straight 22%* 45*	64	5.5 4.1 2.4	7.5 7.0 4.3	8.0 7.5 7.0	8.0 7.5 7.0	98	10.0 7.5 5.0	8.0 7.5 7.0	8.5 6.0 7.0	90 65 73	131	15.0 51.0 7.0	9,0 6,0 7,0	9.5 6.5 7.5	10.0 9.0 7.5	196	27.0 20.0 14.0	10.0 8.5 7.5	10.5 9.0 7.5	11.3 10.0 8,0
12 x 4 (41.6)	Straight 22%* 45*		4.0 4.5 3.0	7.5 7.0 4.5	8,0 7,5 7,0	8.5 7.3 7.0	119	11.0 8.1 5.5	1.0 7.5 7.0	9.0 8.0 7.0	15 15 15 75	159	15.0 11.0 7.0	9,0 8,0 7,0	9.5 6.5 7.5	10,0 9,0 7,5	934	26.0 21.0 14.0	10.0 9.0 7.5	11.0 9.5 8.0	11.5 10.0 8.0
16 x 6 (36.4)	Strolght 22%* 45*	107	42 47 32	8,0 7,0 6,5	8.0 7.3 7.0	8.5 7.5 7.0	161	12.0 9.0 6.0	4.5 8.0 7.0	9.0 8.0 7.0	93 43 75	214	14.0 12.0 8.0	9.5 8.5 7.5	10.0 9.0 7.5	10,5 9,5 8,0	321	30.0 22.0 15.0	11.0 9.5 7.5	11.5 10.0 6.0	12.3 10.5 8.3
10 x 4 (71.5)	Strolght 22%* 43*	135	6.6 5.0 3.2	8.0 7.5 7.9	8.5 7.5 7.0	9.0 8.0 7.5	303	12.0 9.0 8.6	9.0 8.0 7.0	9.5 8.5 7.5	10.0 9.0 7.5	249	17.0 13.0 9.0	13 43 75	10.0 9.0 8.0	11.0 9.5 8.0	403	32.0 24.0 16.0	11,3 9,5 8,0	12.0 10.0 8.5	13.0 11.0 9.0
24 x 6 (86.5)	Straight 2216° 45°	163	7.0 5.1 3.5	80 7.5 7.0	8.5 8.0 7.0	9.0 8.0 7.5	243	13,0 10,0 4,1	9,0 8,0 7,0	9.5 8.5 7.5	10.0 9.0 8.0	334	16.0 13.0 9.0	10.0 8.5 7.5	10.5 9.0 6.0	11.0 10.0 8.5	485	33.0 25.0 17.0	12.0 10.0 8.0	12.5 10.5 8.5	13,0 11,0 9,1
30 x 6 (109.0)	Strolght 2235* 45*	303	7.0 3.4 3.5	8.5 7.5 7.0	8.5 8.0 7.0	9.5 8.0 7.3	304	13.0 10.0 4.3	9,0 8,0 7,5	10.0 9.0 7.5	10.5 9.0 8.0	406	19.0 14.0 10.0	10.0 9.0 7.5	11.0 9.5 8.0	11.5 10.0 8.5	601	34.0 23.0 17.0	12.0 10.0 8.0	12.5 10.5 9.0	13.2 11.3 9,6
34 x 6 (131.3)	Straight 2214* 45*	243	7.1 5.5 3.5	8,5 7,5 7,0	9.0 8.0 7.3	9.5 8.5 7.3	344	13.0 10.0 6.5	9,5 8,5 7,5	10.0 9.0 8.0	10.5 9.5 8.0	490	19,0 14,0 10,0	10.0 9.0 8.0	11.0 9.5 8.0	12.0 10.0 8.5	735	35.0 26.0 18.0	12.0 10.0 8.5	13.0 10.5 9.0	14.0 11.2 9.5
. 14	1 (A)				10.00	100	_	F	FAC	TOIL		_	1000	00-22	1.5%	22.0		5-5-UI			í.
Max Clu Outlet W	ell Area	_	_	0.91		_		_	9.0			-	_	4.0	_				9.4	_	
Min Che, Outlet W	oll Area		_	8,7					97	_	_			4.2		_		_	2.9	_	